

# wb1413

## Multibody Dynamics B

Spring Term 2013, Thu 15:45-17:30, room CT-CZ G, 4 ECTS credits.

### Homework assignment 7

Consider the quick-return mechanism from Figure 1. The crank 2 drives via a slider 3 the rocker 4, and finally the connecting bar 5 moves the slider 6. The centre of mass of link  $i$  is denoted by  $G_i$ . The specification of the mechanism is as follows:  $O_2A = 0.2$  m,  $O_4B = 0.7$  m,  $BC = 0.6$  m,  $O_4O_2 = 0.3$  m,  $O_4G_4 = 0.4$  m,  $BG_5 = 0.3$  m,  $y_c = 0.9$  m,  $m_3 = 0.5$  kg,  $m_4 = 6$  kg,  $m_5 = 4$  kg,  $m_6 = 1$  kg,  $J_4 = 10$  kgm<sup>2</sup>,  $J_5 = 6$  kgm<sup>2</sup>,  $F = 1$  kN,  $T = 0$ . The reduced mass moment of inertia at the balanced crank ( $G_2 = O_2$ ) is  $J_2 = 200$  kgm<sup>2</sup>. The initial angular velocity of the crank is  $\omega_2 = 150$  rpm CCW at  $\theta_2 = 0$  deg. We assume no friction and zero gravity.

Determine the motion of the mechanism by numerical integration of the equations of motion. Derive these equations in a DAE form and stabilize the constraints by means of the Coordinate Projection Method [1]. Try not to derive the equations of motion in an explicit form but evaluate your equations in a step-by-step manner.

Please address the following questions:

- a. Describe your algorithm in words and formula's.

Show for two revolutions of the crank as a function of time:

- b. - The angular speed of crank 2, rocker 4 and connecting bar 5.
- c. - The sliding speed of slider 3 with respect to rocker 4.  
- The horizontal position, speed and acceleration of slider 6.
- d. - The normal force exerted by the slider 3 on the rocker 4.  
- The normal force exerted by slider 6 on the ground.

Finally,

- e. Which checks did you use in order to be sure that you have the correct answers?

Briefly discuss your results.

**Bonus Question:** The motion of this mechanism is clearly periodic, what is the period  $T$  in seconds  $\pm 0.25\%$ .

### References

- [1] Edda Eich-Soellner and Claus Führer. *Numerical Methods in Multibody Dynamics*. European Consortium for Mathematics in Industry. B.G.Teubner, Stuttgart, 1998.

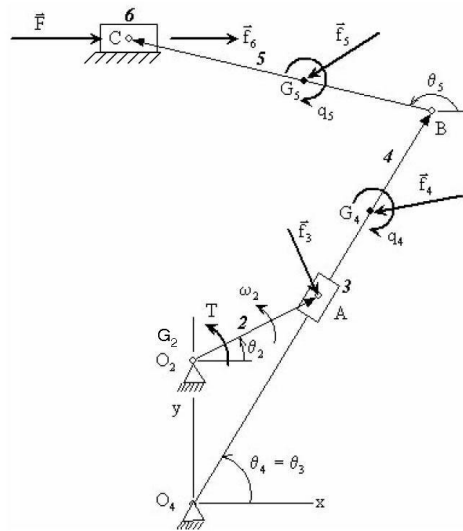


Figure 1 A Quick-Return Mechanism